

## **PROCEDURE**



**PREPARED BY** 



www.hvacsimplified.in



## What is HVAC Load Calculation....?

- □ An <u>HVAC Load Calculation</u> is a mathematical process for measuring several aspects of a building in order to determine the best size, application and style of HVAC system.
- ☐ The purpose is to ensure energy efficiency while also maximizing comfort.
- □ Load calculations have been completed for several years, however, with new technology come quicker and more consistent methods to complete this important step.







## Factors to considering during Cooling load requirement

- □ It is necessary for air conditioning designers to know about the heat source and their nature before designing of air conditioning system.
  - Building detailed layout.
  - \* Building orientation.
  - Ambient condition.
  - Use of Space.
  - Physical dimension of space.
  - Indoor design condition.
  - Filtration level required.
  - Glass area exposed to sun.
  - Occupancy.
  - Lighting load.
  - \* Equipment load.
  - Fresh air requirement.







## **Application of HVAC System**

- Residential
- Commercial
- Industrial
- Educational
- Pharmaceuticals
- Hospitals
- Malls/Offices
- · And many more....











## **Air Conditioning Load Estimate**

- ☐ The air conditioning load is to provide the basis of selecting the conditioning equipment.
- ☐ The load is designed for maximum conditions of heat such as :
- · Peak month.
- Peak day and
- Peak time of the year.



- The design load is divided into two types.
  - Outdoor Load Indoor Load



HUMAN COMFORT







### **OUTDOOR LOADS:**

■ Heat flow through the exterior wall, ceilings, floors, windows and doors due to temperature difference between two sides.

#### LOAD DUE TO SOLAR RADIATION

- ☐ Heat from the Sun absorbed by the wall and roof and later transferred to room by conduction.
- ☐ Heat transmitted directly by radiation through glass of Window.
- ☐ Heat received form infiltration air from windows and doors due to frequent openings.







## **INDOOR LOADS:**

- ☐ People
- ☐ Light
- □ Appliances
- Miscellaneous sources.

People



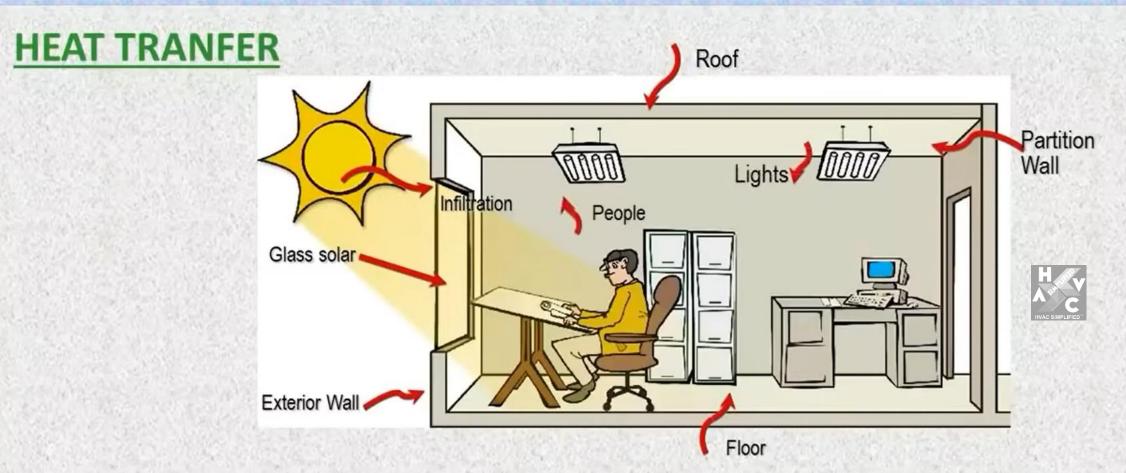
@HVAC\_SIMPLIFIED



Lights







- Heat gain from outdoors through the roof, exterior walls, skylights, and windows. (This includes the effects of the sun shining on these exterior surfaces.)
- Solar radiation heat gain through skylights and windows.
- Conduction heat gain from adjoining spaces through the ceiling, interior partition walls, and floor.
- Internal heat gains due to people, lights, appliances, and equipment in the space.





# EXAPMLE





## E20 FORM

## COOLING **DEHUMIDIFICATION**

#### Table 1-T-100: Cooling and Dehumidifying **Heat Load Estimate Form**

Space	Used for			
Size	×	= Sq.Ft.x		Cu.
Ft.				
Item	Area of	Sun Gain or	Factor	Btu /
149111	quantity	R GAIN GLASS		Hour
_		Sq.Ft.x x	_	-
_	Glass	Sq.Ft.x x	_	_
_	Glass	Sq.Ft.x x	_	
	Glass	Sq.Ft.x x	_	-
Struct and		Sq.Ft.x x	_	_
Sky Ligi		S GAIN - WALLS	ND BOOE	$\rightarrow$
SOLIN	Wall	Sq.Ft.x x	T NOOF	
_	Wall	Sq.Ft.x x	_	-
	Wall.	Sq.Ft.x x		$\rightarrow$
	Wall	Sq.Ft.x x	-	-
	Wall.	Sq.FLx x		
Roof Su		Sq.Ft.x x	-	
Roof \$h		Sq.Ft.x x	-	
		CEPT WALLS AND	PAGE	+
All Glas		Sq. Ft. x x	KOOF	_
Partition		Sq. Ft. x x		_
	1		_	
Ceiling		44.12.	_	
Floor	INFIL TO ATM	Sq. Ft. x x	AUD.	_
				_
Infiltratio		The state of the s	1.08	-
Outside			X 1.08	_
0		ERNAL HEAT	_	
People		People		
Power		H.P. / KW	1.00	$\rightarrow$
Lights	F1-		x 1.08	
Applian			x	+
	ROOM	SENSIBLE HEAT		_
Supply		Supply		_
Duct		Duct		
Heat Go		Leak Loss %		-
'		ROOM SINSIBLE H	EAI	+
to Cites a		LATENT HEAT	0.64	$\rightarrow$
Infiltratio		grib x	83.0	
Outside	Air Clin x		x 0,68	
People		People x		
Steam	6	Lb / hr x 1080		$\rightarrow$
Applian				$\rightarrow$
Vapor T		A Total		$\rightarrow$
Room Latent Heat Sub Total			+	
Supply Leakage		% + SAFETY FAC	TOR %	
		ROOM LATENT HE	AT	
		ROOM TOTAL HE		
Sanchi		x (1-BF) x 1.08	A1	
_				_
Latence :	com x gr/ to Cotal Heat Su	x (1-BF) x 0.68		_

Job Name

	Leak Lo	999			П
TONS = GRAND	TOTAL H	EAT*			
			Peak	Loc	al Tim
Estimated for	Local	Time	Load		Time
CONDITIONS	DB	W8	%RH		Gr/L
Outside			_		
Room					
Difference		xxx	xxx	XX	
Selected Room Co	nditions		B W	6	%R)
SWINGING REVOLVING DOO Open Doors Exhaust Fan Crack SENSIBLE	RS - PE	OPLE X	CFM / F	PERSO R =	N = _
Eff.room sens.Heat				r (ESI	HF)
Eff.room.sens.fleet Eff.room.total heat		Sens H	teat Facto		
Indicated adp		Sens H	feat Facto		°F
Indicated adp	emp. AD	Sens H	feat Facto	umidifi	oF ed rise
Indicated adp	emp. ADI	Sens H	feat Facto	umidifi	oF ed rise









## Q = HEAT FLOW (BTU/HR)

$$Q = U \times A \times \Delta T$$

- U Overall heat transfer coefficient of the material. [U =  $1/\Sigma R$ ]
- A Area of the surface in Square Feet.

@HVAC\_SIMPLIFIED

 $\Delta$  T – Temperature Outside – Temperature Inside.





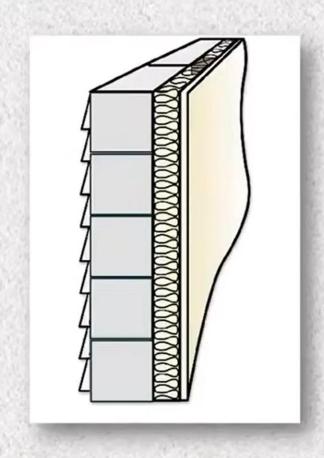
## WALL / ROOF HEAT LOAD

## $Q = U \times A \times \Delta T$

U – Overall heat transfer coefficient of the material. [U =  $1/\Sigma R$ ]

- A Area of the surface in Square Feet.
- $\Delta$  T Temperature Outside Temperature Inside.









## **GLASS HEAT LOAD**

## $Q = A \times SC \times SCL$

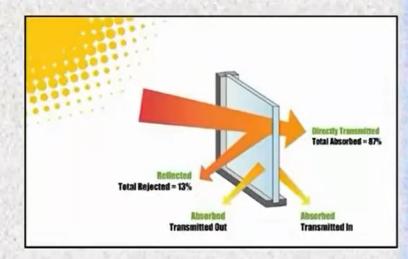
Q = heat gain by solar radiation through glass, Btu/hr [W]

A = total surface area of the glass, ft<sup>2</sup> [m<sup>2</sup>]

SC = shading coefficient of the window, dimensionless

@HVAC\_SIMPLIFIED

SCL = solar cooling load factor, Btu/hr•ft² [W/m²]









- A. SOLAR HEAT GAIN FOR (W) WALL, (F) FLOOR & (R) ROOF.
- B. SOLAR HEAT GAIN FOR (G) GLASS (WINDOWS).
- C. EFFECTIVE ROOM (E.R.S.H.L) SENSIBLE HEAT LOAD.
- D. EFFECTIVE ROOM (E.R.L.H.L) LATENT HEAT LOAD.
- E. EFFECTIVE ROOM (E.R.T.H.L) TOTAL HEAT LOAD.
- F. EFFECTIVE ROOM (E.R.S.H.F) SENSIBLE HEAT FACTOR.
- G. DEHUMIDIFIED (CFM) AIR FLOW.











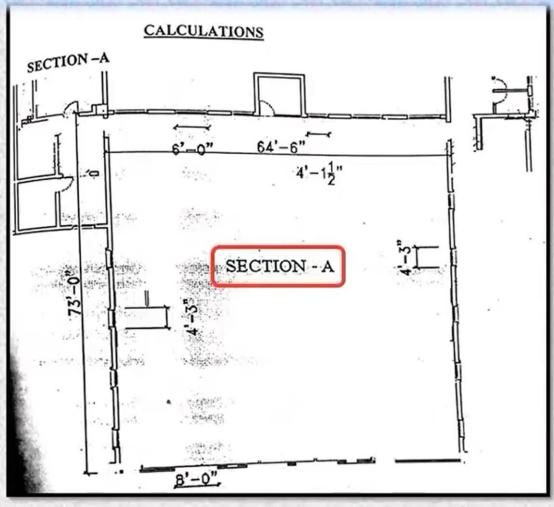
### A. SOLAR HEAT GAIN FOR WALL AND FLOOR

 $Q = U \times A \times \Delta T$ 

Condition Of Wall	Area in Sq.Ft	Temperature Difference in F	U Factor	BTU/Hr
West Partition	730	29	0.36	7621.20
North Exposed	451	31	0.3	4198.95
South Partition	645	29	0.36	6733.80
Floor	4708.5	29	0.19	25943.84
			Total	44497.79



@HVAC\_SIMPLIFIED



#### SECTION - A

Number of Person = 110 Total Floor Area = 4708.5 Sq.Ft







#### A. SOLAR HEAT GAIN FOR WALL AND FLOOR

 $Q = U \times A \times \Delta T$ 

Condition Of Wall	Area in Sq.Ft	Temperature Difference in F	U Factor	BTU/Hr
West Partition	730	29	0.36	7621.20
North Exposed	451.5	31	0.3	4198.95
South Partition	645	29	0.36	6733.80
Floor	4708.5	29	0.19	25943.84
			Total	44497.79

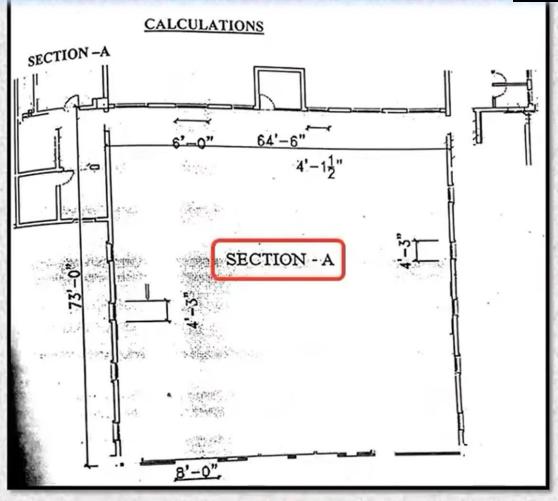
#### **B. SOLAR HEAT GAIN FOR GLASS**

 $Q = A \times SC \times SCL$ 



@HVAC\_SIMPLIFIED

Condition Of Wall	Area in Sq.Ft	sc	SCL	BTU/Hr
North Exposed	193.5	0.8	16.08	2489.28
			Total	2489.28



#### SECTION - A

Number of Person = 110
Total Floor Area = 4708.5 Sq.Ft







H

#### **HEAT LOAD CALCULATION**

#### C. EFFECTIVE ROOM SENSIBLE HEAT LOAD (E.R.S.H.L)

1. Outside Air	Volume of the room (ft3) x No. of air changes	= 4708.5 x 10 x 2	=1569.5 CFM
1. Outside Air	60	60	

$$= 110 \times 245$$

$$= 4708.5 \times 2 \times 3.4$$

$$= 90 \times 300 \times 3.4$$

@HVAC\_SIMPLIFIED



#### Where:

13.5 = Specific Volume of moist air at 70F DBT,

0.224 = Specific heat of moist air at 70F and 50%RH

and

0.15 = bf

245 = Sensible Heat Contribution in Room (ASHRAE)











#### D. EFFECTIVE ROOM LATENT HEAT LOAD (E.R.L.H.L)

1. QL (Latent Heat)

= Outside air CFM x (Δw) x (bf) x 0.68

 $= 1569.5 \times 36 \times 0.15 \times 0.68$ 

= 5,763.204 btu/hr.

2. Latent Heat Internal Load

= No. of People x Latent Heat Gain.

 $= 110 \times 205$ 

= 22,550 btu/hr

3. Total Latent Heat Load (1+2)

@HVAC.SIMPLIFIED

= 5,763.204 + 22,550 = 28,313.204 btu/hr Safety Factor 8%

= 30,578 btu/hr 4. Effective Room Latent Heat Load





(Δw) = Outside Humidity Ration - Indoor Humidity Ration

= Latent Heat Contribution in Room (ASHRAE)

0.15

205

@HVAC.SIMPLIFIED

= bf



#### E. EFFECTIVE ROOM TOTAL HEAT LOAD (E.R.T.H.L)

E.R.T.H.L = E.R.S.H.L + E.R.L.H.L

= 222,914.87 + 30,578

= 253,492.87 btu/hr

TR = E.R.T.H.L / 12,000

= 253,492.87 / 12000

= 21.12 TR (22 TR approx.)

E.R.S.H.L 222,914.87 **EFFECTIVE ROOM** = 0.88SENSIBLE HEAT FACTOR E.R.T.H.L 253,492.87

E.R.S.H.L 222,914.87 DEHUMIDIFIED (C.F.M) 11466.81 CFM 1.08 x (RT - ADP) 1.08 x (72 - 54)

@HVAC\_SIMPLIFIED





www.hvacsimplified.in



#### E. EFFECTIVE ROOM TOTAL HEAT LOAD (E.R.T.H.L)

E.R.T.H.L = E.R.S.H.L + E.R.L.H.L

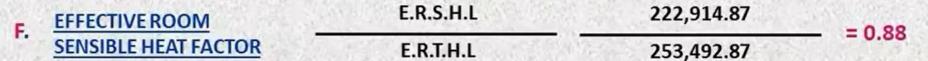
= 222,914.87 + 30,578

= 253,492.87 btu/hr

TR = E.R.T.H.L / 12,000

= 253,492.87 / 12000

= 21.12 TR (22 TR approx.)



G. <u>DEHUMIDIFIED</u> (C.F.M)

E.R.S.H.L

222,914.87

1.08 x (RT – ADP)

1.08 x (72 – 54)

1.08 x (72 – 54)

Capacity: 22 TR

Air Flow: 11467 CFM







